

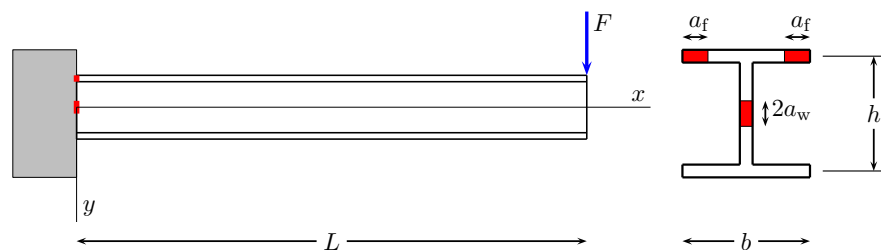
RAK-33060 Fracture mechanics and fatigue

2. Exercise

1. A tip loaded cantilever I-beam has cracks at the clamped end. The cross-section can be considered as an ideal I-profile. In the web there is a crack of length $2a_w = 2t$ positioned symmetrically about the neutral axis. At the flange tips there are two symmetrically positioned cracks. How long ($a_f = ?$) these flange tip cracks should be in order to be more dangerous than the crack in the web? Thickness of the web is t and the flanges $3t/2$, respectively. The other dimensions are related as $L/h = 10$, $h/t = 50$ and $b = h/2$. The fracture toughness in the mode II is $K_{IIc} = (\sqrt{3}/2)K_{Ic}$, where K_{Ic} is the mode I fracture toughness.

You can assume that the shear stresses are distributed uniformly in the web. As the cross-section is assumed to be an ideal I-section, the moment of inertia for the web can be neglected. The bending stresses can also be assumed to have a constant value in the flanges.

Tables of stress intensity factors are at the end of this paper.



2. Determine the stress intensity factor K_I for a penny-shaped crack of radius a in an infinite domain under uniaxial stress σ . Use the Griffith energy approach assuming that stresses are relaxed in a ball of radius a around the crack. Compare to the values you have found in the literature.

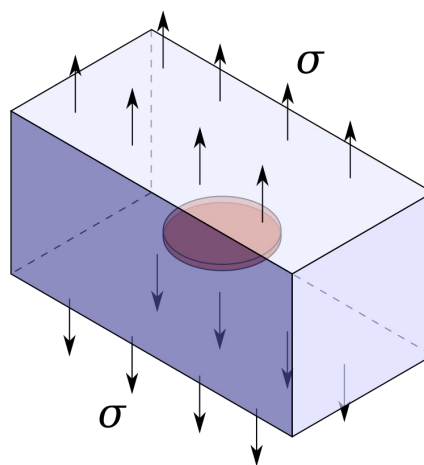
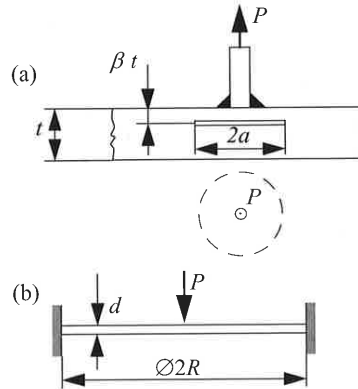


Figure: Bbanerje - Own work, CC BY-SA 3.0, <https://commons.wikimedia.org/wiki/User:Bbanerje>

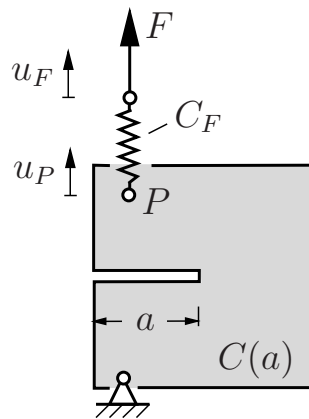
3. A thick plate containing a circular delamination crack is loaded by a point force according to figure (a). Determine the stress intensity factor K_I and decide the critical load for fracture if $K_{Ic} = 200 \text{ MPam}^{1/2}$, $\beta = 0.1$, $t = 10 \text{ cm}$, $a = 20 \text{ cm}$.

Hint. For a circular rigidly fixed plate according to figure (b) the displacement of the loading point due to a force is

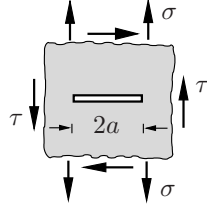
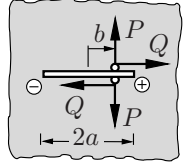
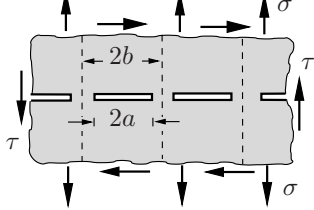
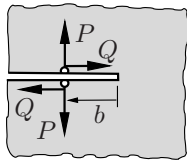
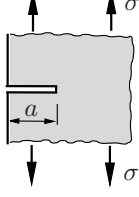
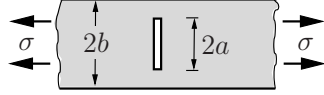
$$\Delta = \frac{3(1 - \nu^2)}{4\pi} \frac{PR^2}{Ed^3}.$$



4. Consider a DCB-test (Double Cantilever Beam) in a flexible testing apparatus. Draw the dimensionless crack driving force $\mathcal{G}/\mathcal{G}_c$ as a function of the dimensionless crack length a/a_0 using different values of the flexibility ratio $C_F/C(a_0)$. How flexible has the testing machine to be to produce unstable crack growth for a material which has a shallow (nearly constant) R -curve?



Gross, Seelig: Fracture Mechanics, figure 4.44.

| | | |
|---|---|--|
| 1 |  | $\begin{Bmatrix} K_I \\ K_{II} \end{Bmatrix} = \begin{Bmatrix} \sigma \\ \tau \end{Bmatrix} \sqrt{\pi a}$ |
| 2 |  | $\begin{Bmatrix} K_I^\pm \\ K_{II}^\pm \end{Bmatrix} = \begin{Bmatrix} P \\ Q \end{Bmatrix} \frac{1}{\sqrt{\pi a}} \sqrt{\frac{a \pm b}{a \mp b}}$ |
| 3 |  | $\begin{Bmatrix} K_I \\ K_{II} \end{Bmatrix} = \begin{Bmatrix} \sigma \\ \tau \end{Bmatrix} \sqrt{2b \tan \frac{\pi a}{2b}}$ |
| 4 |  | $\begin{Bmatrix} K_I \\ K_{II} \end{Bmatrix} = \begin{Bmatrix} P \\ Q \end{Bmatrix} \frac{2}{\sqrt{2\pi b}}$ |
| 5 |  | $K_I = 1.1215 \sigma \sqrt{\pi a}$ |
| 6 |  | $K_I = \sigma \sqrt{\pi a} F_I(a/b)$ $F_I = \frac{1 - 0.025(a/b)^2 + 0.06(a/b)^4}{\sqrt{\cos(\pi a/2b)}}$ |