



# Development of a quasi-steady-state assessment method of night cooling

H. Breesch<sup>1</sup>

K. Goethals<sup>2</sup>, A. Janssens<sup>2</sup>

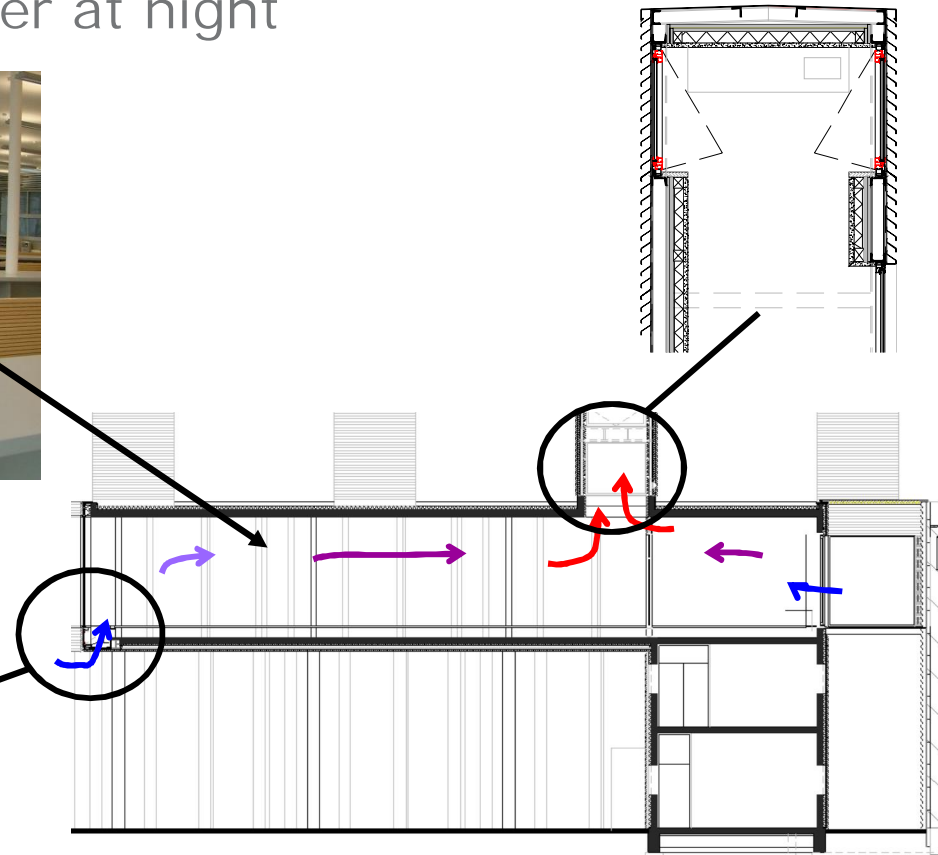
<sup>1</sup> Catholic University College Ghent, Belgium

<sup>2</sup> Ghent University, Belgium



# Night cooling

- Passive cooling technique
  - Accumulation heat gains in thermal mass by day
  - Convective heat transfer at night



Renson (Belgium)


# Context

## Assessment method night cooling?

- Directive on the Energy Performance of Buildings 2002/91/EC (EPBD)
- EN 13790: quasi-steady-state < > detailed BES

⇒ new quasi-steady-state assessment method in Flemish EPB

- cooling demand
- mechanical night cooling
- non-residential buildings

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- Introduction
  - **Methods**
    - Quasi-steady-state calculation
    - Type buildings
  - Results
  - Conclusions & further research

# Quasi-steady-state calculation

- EN 13790

- Cooling demand (MJ)  $Q_{C,nd} = (Q_{int} + Q_{sol}) - \eta_{C,ls} (Q_{tr} + Q_{ve})$

- Heat transfer by ventilation  $Q_{ve} = H_{ve,adj} (\theta_{int,set,C,z} - \theta_e) t$

$$H_{ve,adj} = \rho_a c_a \sum_k b_{ve,k} q_{ve,k,mn}$$

- Temperature adjustment factor

$$b_{ve,k} = \frac{(\theta_{int,set} - \theta_{sup,k})}{(\theta_{int,set} - \theta_e)}$$

- Extra air flow rate

$$q_{ve,extra,mn} = c_{ve,eff,extra} f_{ve,t,extra} q_{ve,extra}$$

- Adjustment factor inertia

- Time fraction of operation

ASSOCIATIE  
K.U. LEUVEN

=> Define adjustment factors

# Type buildings

- Dynamic simulations in 3 non-residential buildings

- Small office building: 3 floors

1. Cafeteria /offices

2. Lobby/offices

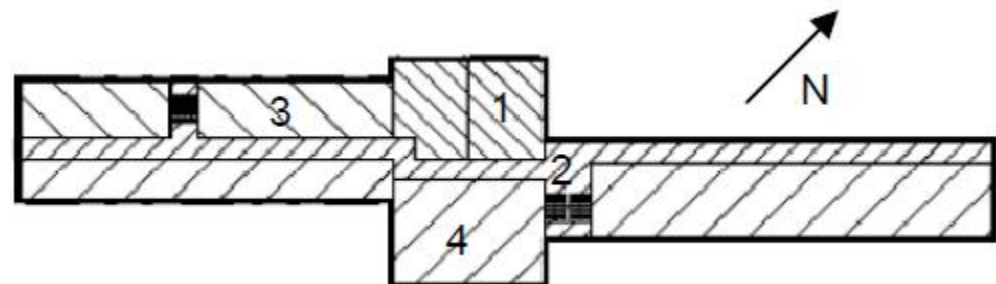
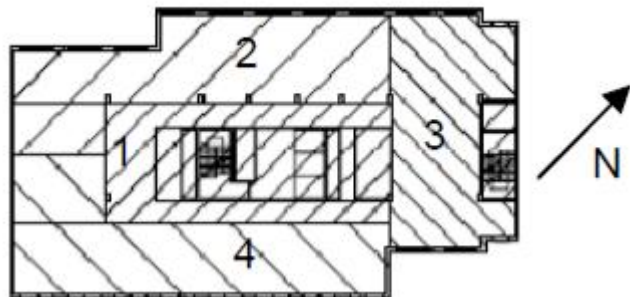
3. Technical rooms  
/meeting room

4. Sanitary



- Large office building (10 floors) – school building (1 floor)

- 3 quality levels: ACcEptable – GOOo – VERy good



# Type buildings

- Internal heat gains: 12.5 – 100% default EPB
- Meteonorm Uccle (Belgium)  $\approx$  weather 2004-2008
- Balanced mechanical hygienic ventilation
- Convective heating ( $<20^{\circ}\text{C}$ ) and cooling ( $>23^{\circ}\text{C}$ )
- Night cooling
  - cafeteria, lobby, offices, meeting rooms, class rooms
  - Flow rate: 1 – 3 – 6 x hygienic flow rate
  - Control system


Daytime activation requirement	Night cooling activation requirements
$\theta_{i,\max} > 23^{\circ}\text{C}$	$22\text{h} < \text{time} < 6\text{h}$ $\theta_{s,\text{ceiling}} > 22^{\circ}\text{C}$ (= heating set point + $1^{\circ}\text{C}$ ) $\theta_i - \theta_e > 2^{\circ}\text{C}$ $v < 7 \text{ m/s}$

$\theta_i$  : indoor temperature ( $^{\circ}\text{C}$ )

$\theta_s$  : surface temperature ( $^{\circ}\text{C}$ )

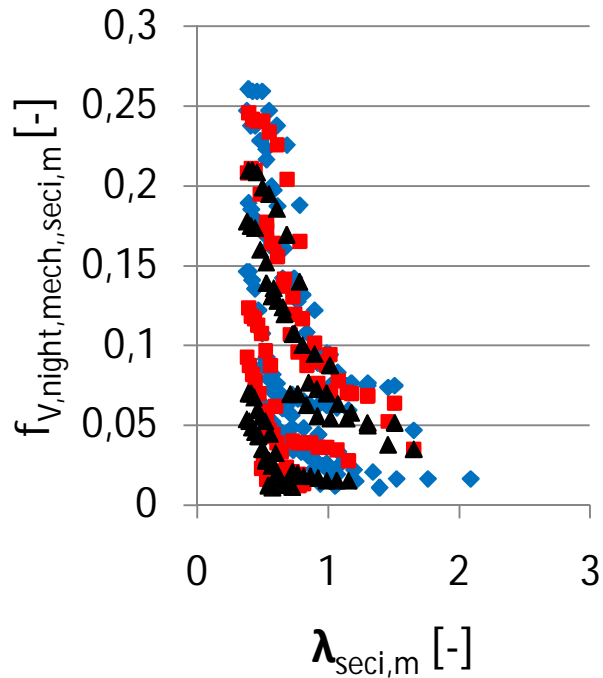
$\theta_e$  : exterior temperature ( $^{\circ}\text{C}$ )

$v$ : wind velocity (m/s)

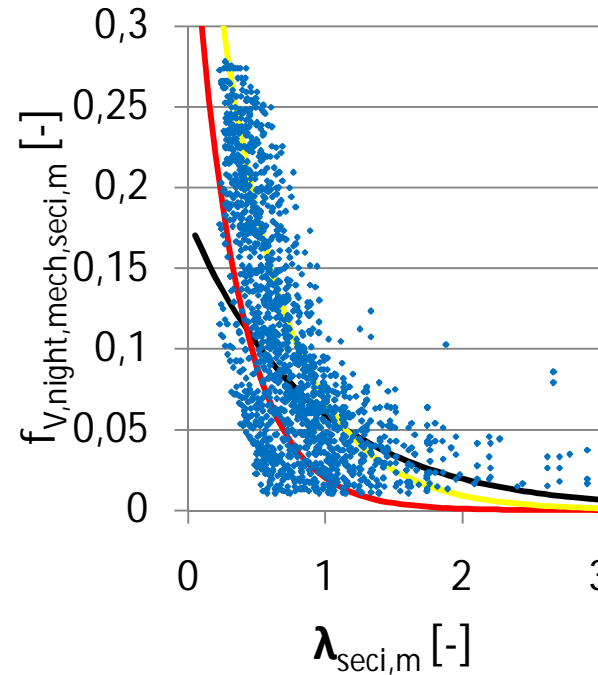
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- Introduction
  - Methods
  - Results
    - Determination of adjustment factors
    - Verification
  - Conclusions & further research



# Time fraction of operation f



◆  $V_{\text{night,mech1,cool/overh,sec i}}$   
 ■  $V_{\text{night,mech2,cool/overh,sec i}}$   
 ▲  $V_{\text{night,mech3,cool/overh,sec i}}$



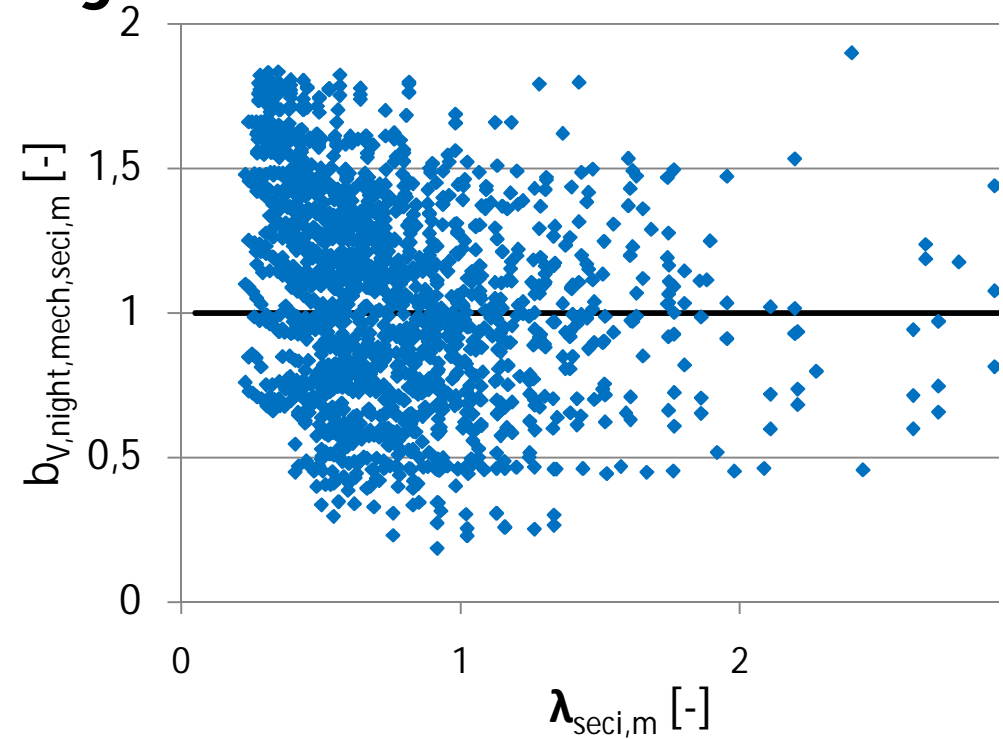
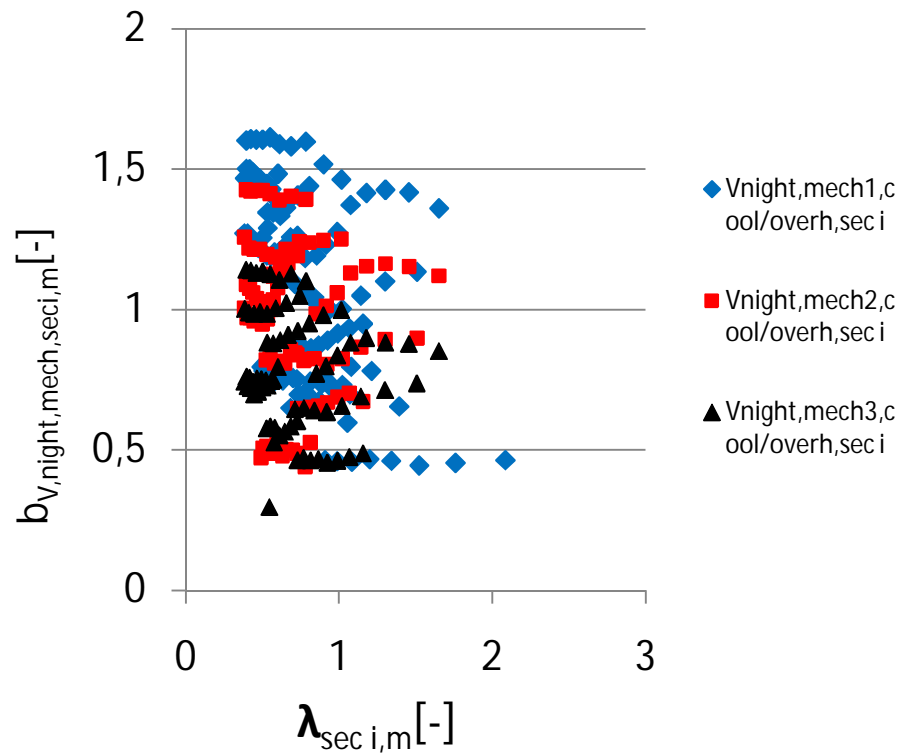
—  $k=0.18$  en  $l=-1.11$   
 —  $k=0.4$  en  $l=-3$   
 —  $k=0.5$  en  $l=-2$

- Function of losses to gains ratio  $\lambda$

$$f_{V,\text{night,mech,overh/cool,seci,m}} = k \cdot e^{l \cdot \lambda_{\text{seci,m}}}$$

Best fit:  $k = 0.18 - l = -1.11$

# Temperature adjustment factor b



- No significant relation  $b - \lambda$
- Temperature adjustment factor  $b = 1$

# Adjustment factor inertia c

- 4 levels of thermal capacity
- Saved cooling demand compared to base case

	$D_j$ (kJ/m <sup>2</sup> .K)	Saved cooling demand (% base case)		
		ACC	GOO	VER
Lowered ceiling + raised floor + heavy façade	60	0.63	0.66	0.69
No lowered ceiling + raised floor + light façade	194	0.64	0.71	0.76
No lowered ceiling + raised floor + heavy façade	256	0.96	0.93	0.92
No lowered ceiling + no raised floor + heavy façade	366	1.00	1.00	1.00

- Adjustment factor inertia



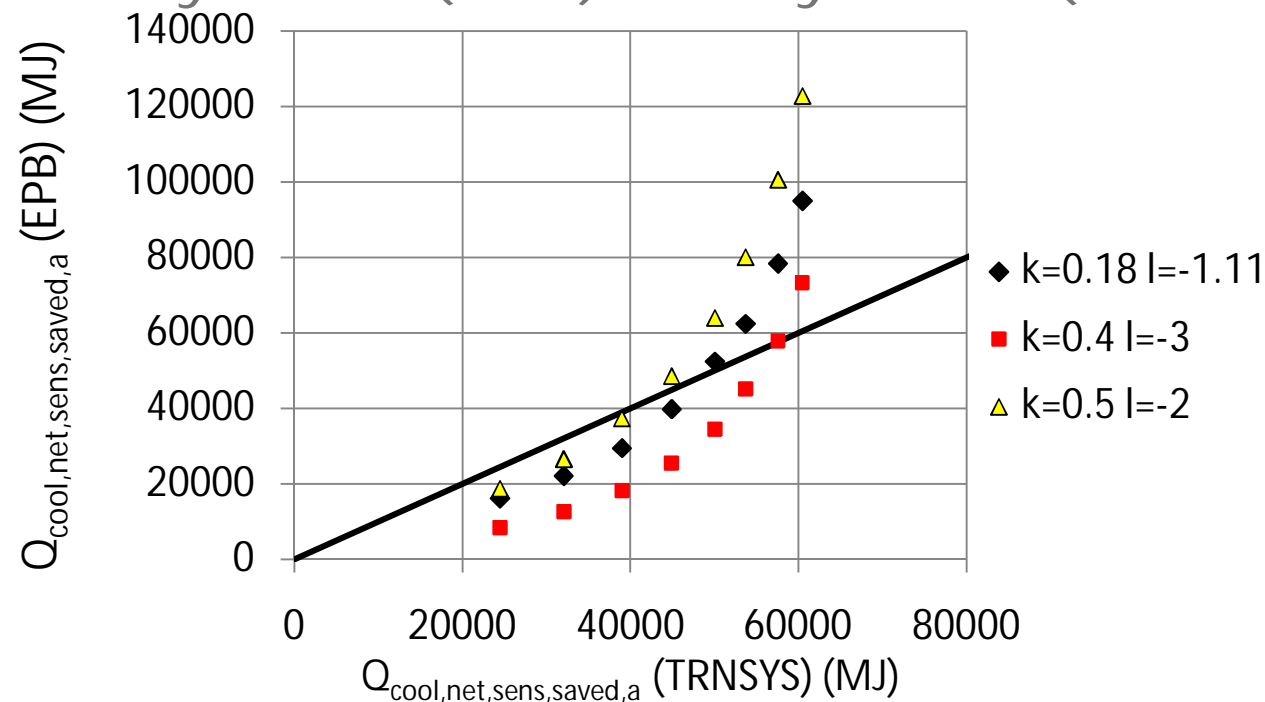
- Raised floor: **light** – **heavy** structure: **0.7** – **1.0**
- Lowered ceiling: 0.7



# verification

- Saved cooling demand

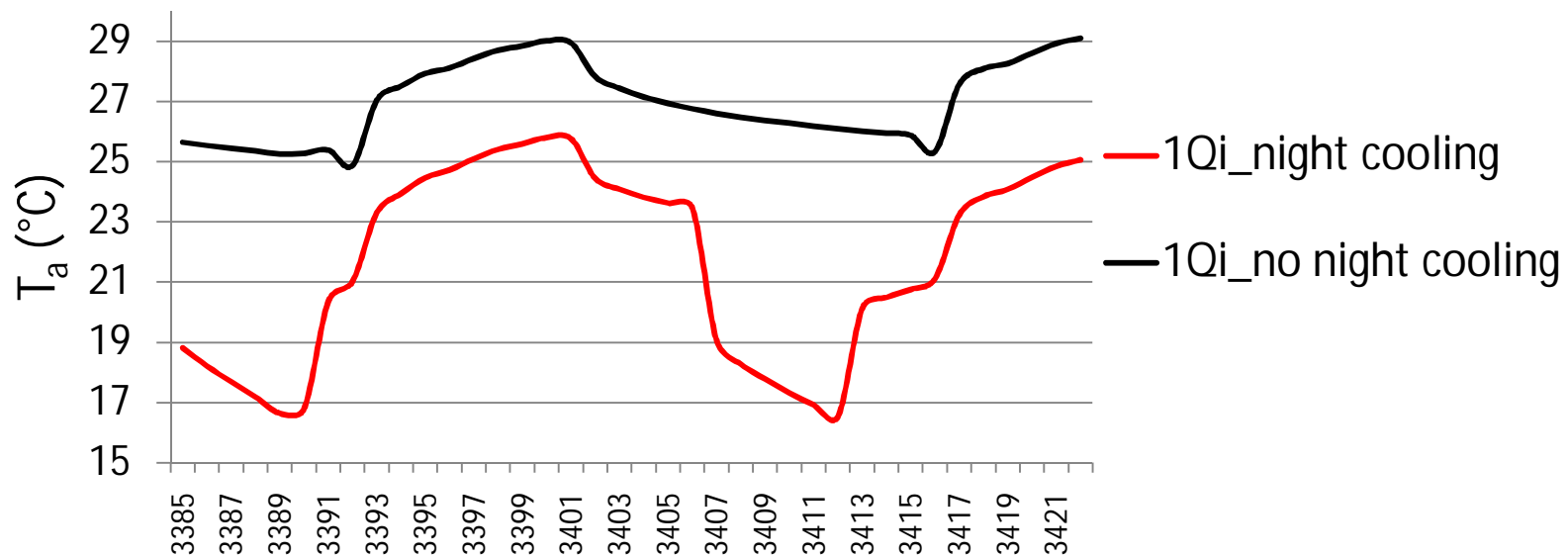
Quasi-steady-state (EPB) < > dynamic (TRNSYS)



- $k=0.18$ ,  $l=-1.11$  overestimation high internal heat gains
- Time fraction of operation:  $k=0.4$ ,  $l=-3$

# verification

- Uncertain correspondence EPB < > dynamic
  - Difference thermal storage
- EN 13790 to assess night cooling questioned
  - Standard utilization factor  $\eta$
  - Ventilation heat transfer coefficient  
extra flow rate -> temperature course is changed



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# Conclusions & further research

- New quasi-steady-state assessment method of mechanical night cooling
- EN 13790: ventilation heat transfer coefficient
  - Temperature adjustment factor = 1
  - Time fraction of operation ~ losses to gains ratio
  - Adjustment factor dynamic effects ~ thermal mass
- Dynamic simulations 3 non-residential buildings
- Quasi-steady-state < > dynamic
  - Non-linear & uncertain correspondence
  - Dynamic effects insufficiently considered
- Further research to assess effect of night cooling